



3

$$A_{nj} = \frac{w_b G \sqrt{C_{nl}}}{b N} = \frac{c_b 4 \epsilon}{h_b G \theta_j} H$$



3 Angular contact ball bearings



G-w nwn dfufnbw		Egt\Snbl\ont\dtlg5	α	9
Single row angular contact ball bearings	385			
Basic design bearings.....	385	i sof JN dbc 3t		
Bearings for universal matching	385	47 Single row angular contact ball bearings		406
Bearings with 25° contact angle (AC series)	386	47 Double row angular contact ball bearings.....		424
Double row angular contact ball bearings	386	47 Capped double row angular contact ball bearings .		428
Basic design bearings.....	386	47 Four-point contact ball bearings.....		430
Bearings with a two-piece inner ring.....	386			
Four-point contact ball bearings	387			
Bearings with locating slots	387			
SKF Explorer bearings	387			
Capped bearings	388			
Greases for capped bearings	38			
Grease life for capped bearings	38			
Cages.....	3 0			
ohbsnj -Pbu				
(Dimension standards, tolerances, contact angle, internal clearance, preload, permissible misalignment)				
mog6a	4 •			
(Minimum load, equivalent dynamic bearing load, equivalent static bearing load)				
Calculating the axial load for bearings mounted singly or paired in tandem.....	400			
Load carrying capacity of bearing pairs	400			
Vg5 Dgsbl Jsgd5 Vtα	α			9
i gs5 Vt V 3α Dggf α	α			9
Egt\SnobntV gsbl\ontα	α			94
Single row angular contact ball bearings.....	403	• b -ufn c fuTpnbf Tbhf h-fun w		
Proper adjustment	403	Db5 C4H =====		31
Axial loads in one direction.....	403	Bearings with Solid Oil		1023
Load ratio	403	NoWear coated bearings		105
Four-point contact ball bearings	403	Super-precision bearings		→ skf.com/super-precision
Used as a thrust bearing.....	403	Hybrid bearings		→ skf.com/super-precision
Load ratio	403			

ychh-fun cnbw

v wc εhbGhij vnnWHfv o 3G#CG#Thεwl C5 C
l UHεhPwH GdεbGhε bHHPεnεPCwc 4εCεx =
bnj w4G#CnlnBεε b4εhbGhij HεVThdεTbKh=
5 bPhεbnεbqqGhNbc 4ε#Cnln GdεwLcnεCεTh=
bNTUkh5 hnlεCεε C GhεC5 qbNεx hLjTI =
HbwThi .ePt Tnt .t T9 D6T Pe.xttP9 y0Mεne.
PnSxnzPe.sP6ky 6EM
E Pux TPe. Thos9 xuBn.xyovuuεPt P.Dsoe
vzvt .xne.vxs Tknut .fos. Thevt us Tε6x DD6εx
uBnt .zxn.yP.tv DD6Pe.on.sPr v Pt u

o-fun -fbcu-w

ATP pfb-Tp hn- pf w

. AL Tε66xet .Th.onPe TεPzUbn.onMfos.
t Thi εsoK.yPxsThi t
. AL Tε66xet .Th.PTSPSe TεPzUbn.fos.eovy εP.
soK.xne.fovs DoTuzonuxzuyPxsThi t

y pf Tfuj n TfFtTlj

. l SP.εKPs.t Sov εPs.Pnxy εT.x.6si P.
nv9 yPs.ohyx ε.ω.yP.ThzosDoxuPe.Th.
t Thi εsoK.yPxsThi t .i TWhi .uSP9 .uSPε.
sPεuTWPST S.εxe.zxssMThi .zxDxzTM
. BPzxtvP.ohuεP.t Pzone.soK.ohyx ε .x.
6si P.nv9 yPs.ohyx ε.xsP.ThzosDoxuPe.
Th.eovy εsoK.yPxsThi t .i TWhi .uSP9 .
uSPε.ST S.εxe.zxssMThi .zxDxzTM
. A.6si P.nv9 yPs.ohyx ε.xsP.ThzosDoxuPe.
Th.fovs DoTuzonuxzuyPxsThi t .i TWhi .
uSP9 .uSPε.ST S.εxe.zxssMThi .zxDxzTM

app ucnn n FupF-ub-w

l T S.tDPPet .sxDE.xzzPεsuBnt.xne.
ePzPεsuBnt.xsP.Dott Tj εP

El u BuyBS bys yBwu

B l sCL yB Kys gCBwygwey elys B u

kbGt Thi εsoK.xni v εs.zonuxzuyx εyPxsThi t.
.zxn.xzoz9 9 oexuP.xL Tε66xet .Th.onP.
e TεPzUbn.onM ST .uMDP.ohyPxsThi .Tε.uMDEx εM
xelvtuPe.xi xTh ux.t Pzone.yPxsThi .l SPε.
yPxsThi .sThi t.SxwP.xn.vDDPs.xne.x.εKPs.
t Sov εPs.xne.xsP.non tPDxsxy εP

hwabgB. gv. gaaDvb 2Bb

. yPxsThi t .Th.uSP. C J l B 3 B(E) series
with 40° contact angle
9 some sizes in the 70 B series
9 sealed bearings:
in series 72 B(E) (15 ≤ d ≤ 55 mm)
in series 73 B(E) (12 ≤ d ≤ 50 mm)
9 bearings in the 72 AC series with 25°
contact angle (15 ≤ d ≤ 70 mm)
9 bearings in the 73 AC series with 25°
contact angle (17 ≤ d ≤ 70 mm)
9 some large size bearings with a flanged
outer ring (skf.com/go/17000-3-1)
9 SKF inch bearings (ALS and AMS series,
skf.com/go/17000-3-1)

oga .2a BS2gv B a

- 9 are intended for adjusted arrangements where only one bearing is used at each bearing position and are not suitable for mounting immediately adjacent to each other
- 9 have Normal tolerances on bearing width and stand-out of the rings
- 9 have different performance capabilities compared with SKF Explorer bearings

o2gv B a DvdB e2vag gb B

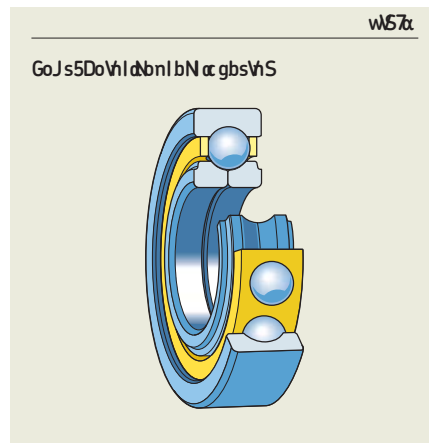
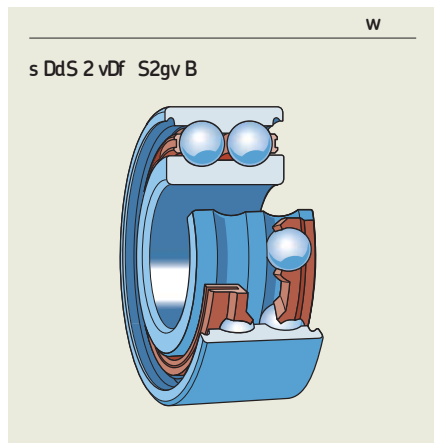
- 9 are available with 25° and 40° contact angles
- 9 are intended to be used in sets
- 9 have ring widths and standouts manufactured to tight tolerances
- 9 can also be used in place of basic design bearings for arrangements with single bearings, as they typically have higher precision, and increased load carrying capacity and speed capability

When two bearings are mounted immediately adjacent to each other, a given internal clearance or preload or an even load distribution between the two bearings is obtained without the use of shims or similar devices.

Bearings for universal matching are identified by the following suffixes:

- 9 CA, CB, CC or G for internal clearance
- 9 GA, GB or GC for preload

When ordering, indicate the number of individual bearings required and not the number of sets.



i a pendn

SblthP5 Cwnlthj hbnεhPCnhthTGHx bdl+
.3j 1 Q

-3bnPh8 -bssbnj h8 hnu

6=WhhP× ThGhThCbPnbGdlthj hbnqbnWd=
Ciεhthj hεhbGdhj hthbPhFwblh=
6=ThbGhThGpW4bnPbyW4CbPhhFw4d=
6=TbHqbg4h4CbP4thH=
6=NbnθNc5 5 CPblhbyW4CbPhhεCnh=
PWhNWhCn4d=
a=byW4CbPhhNthεCITPWhNWhhε=
ITWhεhbGdhj εPwhhPdhj bthHTh=
hbnPh5 εqblG5 wHεhεPPhP

-o bN4UyOc bN4-bssbnj h8 hnu

6=qGCKWhhεh4lWh4dHwεhbGdhj =
bGbnj h5 hnl=
6=NbnθNc5 5 CPblh4Udhj 5 C5 hnlH=
6=TbH4CbP4thHThblPWhG hεG5 Th=
chbGdhj εyW=
6=NbnθNc5 5 CPblhbyW4CbPhhεCIT=
PWhNWhhεwεCn4dεCnhεhbGdhj εh=
hbNTPWhNWh

-Gbn4UyOibN4-bssbnj h8 hnu

6=WhhHhnhHWhhεC5 Wb4jn5 hnlεwεC=
bHhWdhεbN2 IC cbN2εGbnj h5 hnl=
6=TbH4CbP4thHThblCnKhG hεCx bGPhTh=
chbGdhj εyW=
6=NbnθNc5 5 CPblhbyW4CbPhhεCIT=
PWhNWhhεwεCn4dεCnhεhbGdhj εh=
hbNTPWhNWh

o hbsVhStα Vka 11d bnlbNα
bnS3αApd gsvgt-

=TbKhεGbnx bdj hC5 hGdεCqlU5 WhPεCG=
TjT+qhhPH
=TbKhεGpWnhP+hnhHWhdεCbyW4CbPdhj =
bnP5 Wb4jn5 hnlεwεCn4dεCnhεhbGdhj =
bNc5 5 CPblhεx CεU5 hHThThG5 qbN=
CbPhεhiCGhPj hHGHhHhbnεCnwG
εGhεFwdqhpε WTεnεCqlU5 WhP5 bNThhP=
cGhHhbj hεHhbnPbP

DC5 qbGhP× WTεhbGdhj Hx WT40° contact angle, benefits include:

- 9 20% higher limiting speeds
- 9 higher radial load carrying capacity (by trading off lower axial load carrying capacity)
- 9 increased robustness when used as the backup bearing in sets that are predominantly loaded in one direction

ECKe l sCL yB Kys
gCBwygwey el ys B u

The design of SKF double row angular contact ball bearings (. 57α, DbSgα•1) corresponds to two single row angular contact ball bearings arranged back-to-back, but takes up less axial space. They can accommodate radial loads, axial loads in either direction and tilting moments. Double row angular contact ball bearings provide stiff bearing arrangements.

mLGdlbnf bsf dttosl5 gnl

- 9 bearings in the 32 A and 33 A series
- 9 bearings with a two-piece inner ring
- 9 capped bearings
- 9 open bearings (that are also available capped) that may have recesses in the ring side faces

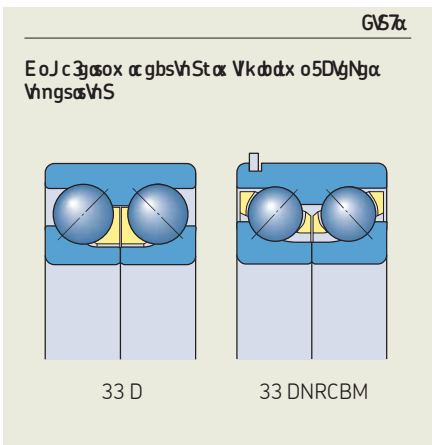
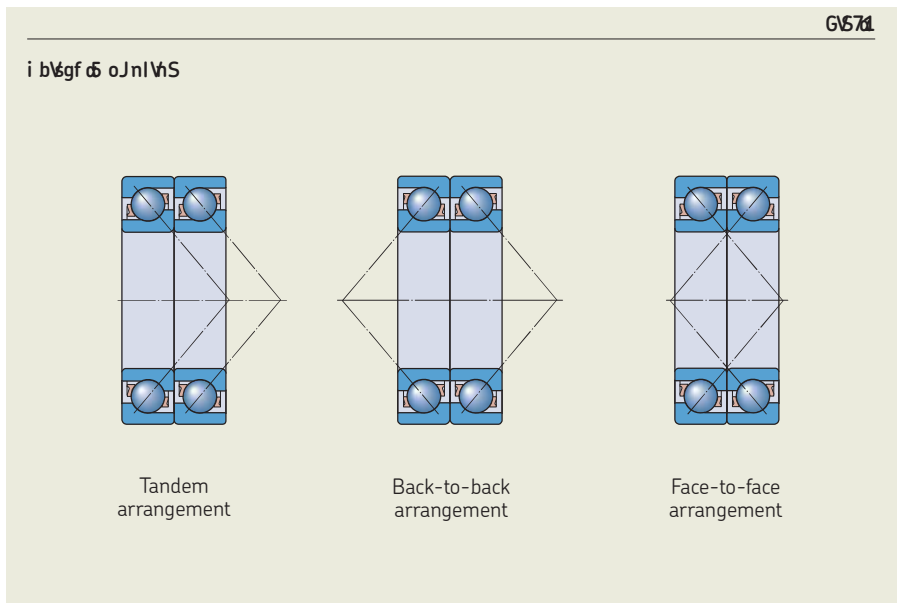
Bearings in the 52 and 53 series are no longer available and have been replaced with 32 A and 33 A series bearings, which are dimensionally interchangeable. Only size 3200 A is different, and has a width of 14 mm instead of 14,3 mm.

Bbt Wf gt V5nα gbsVhSt

- 9 have different tolerances and performance capabilities compared with SKF Explorer bearings

BgbsVhStα Vkdδx o5DyNga
VhngsαVhS

- 9 incorporate a larger number of balls, and have a larger contact angle, giving the bearing its high load carrying capacity, especially in the axial direction
- 9 are separable in the 33 D series (. 57α), i.e. the outer ring with ball and cage assemblies can be mounted independently of the inner ring halves
- 9 are non-separable in the 33 DNRCBM series (. 57α)
- 3 have a snap ring groove with a snap ring in the outer ring, enabling simple and space-saving axial location in the housing
- 3 have been designed specifically for centrifugal pumps, but can also be used in other applications

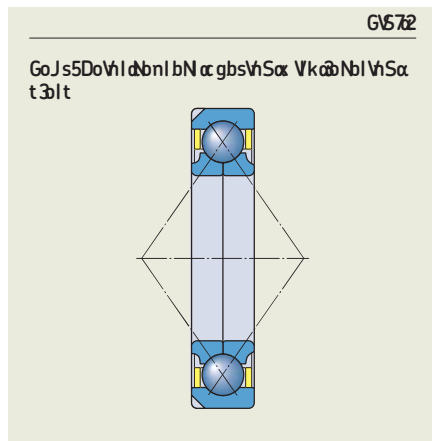


tCKs DC BwgCBwgyw ey elys B u

tCwG qCbl#CnlbN#b4#hbGuj H# . **S7x** =
DbSgα•1 QbG#Pb4#uj 4#Cx #nj w#G#Cn
 lbn#b4#hbGuj H# WT#bN# bdH#Tbl #G#
 PhHjnhP#C#wqCG#byl#4CbPH#CIT=
 P#NlCnH#CG#; W#n#byl#4CbP#b45 W#P=
 G#P#4CbP#bn#b4C#h#wqCG#hP#gCdl udK#
DbSgα 94 Q#W#h#hbGuj H#G#hqbG#C4#L#h =
 lTh#W#hG#uj # WT#b4#bnP#bj h#H#5 c4#
 Nbn#h#5 CwnlH#P#hqbG#lH#4#G5 #Th#x C=
 W#h#hbGuj H#b2h#wq#CnH#PhG#C4#H#H#
 byl#4CbN#Tbn#P#Cw#4#Cx #hbGuj H#
 oCIT#h#hbGuj #Tb4#h#C# Lt #yq4CG#
 iCwG qCbl#CnlbN#b4#hbGuj H#bK#h#
 GhN#H#P#T#Cw#PhG#W#H#5 q#CK, ar rg
 g , ob , S, Q#o#a ca, (om SoQbro
 g b Q# hwn# o(J nQ Jr , JS, Q#o# . **S7x**
DbSgα 94 d Q((bro b , a , J, #aa, a nQ
 S, ca, (br -Om bQ# (a rcobo#
 , o r#J sr obm#ob#n#SQ S, Q#o#a QJ,
 acS, n# (br • mQ s#o# r#J#ab , J oo, J
 Jo#(, r#J Qbro a b(

mL Gd lbnf bsf dttos l5 gnl

S, Q#o#a ob , f lB i 3 series
 9 some sizes in the QJ 10 and QJ 12 series
skf.com/go/17000-3-4



BgbsVhStα VkaNblVhSα 3lt

Four-point contact ball bearings can be supplied with two locating slots in the outer ring (designation suffix N2, . **S7x**):

- 9 preventing the bearing from turning
- 9 positioned 180° apart

The dimensions and tolerances of the locating slots are in accordance with ISO 20515 and are listed in [lbc 3α](#).

ct FdD Csl s elys B u

For information, refer to [DbSgα](#)



WbC 3α

MbNblVhSα 3lt d#dkg#J lgs#VhS#R#J s5DoVhIdbnlbnα b3#x gbsVhSt

hJ l t V g d V b 5 glgs		E V g n t V n t Diameter series 2			Diameter series 3			W b 3 s b n N g ¹⁾
D	≤	h	b	r ₀	h	b	r ₀	t U
mm		mm						mm
79	89	2,5	3,5	0,5	—	—	—	0,2
89	: 0	3	4,5	0,5	3,5	4,5	0,5	0,2
: 0	(3,5	4,5	0,5	3,5	4,5	0,5	0,2
(fi9	4	5,5	0,5	4	5,5	0,5	0,2
fi9	669	5	6,5	0,5	5	6,5	0,5	0,2
669	670	6,5	6,5	0,5	8,1	6,5	1	0,2
670	689	8,1	6,5	1	8,1	6,5	1	0,2
689	6(0	8,1	6,5	1	10,1	8,5	2	0,2
6(0	6fi0	10,1	8,5	2	11,7	10,5	2	0,2
6fi0	60	10,1	8,5	2	11,7	10,5	2	0,2
60	80	11,7	10,5	2	11,7	10,5	2	0,2
80	(0	11,7	10,5	2	11,7	10,5	2	0,2
(0	800	12,7	10,5	2	12,7	10,5	2	0,4

¹⁾ Other tolerances are in accordance with ISO 20515.

pyDDI S el ys B u

kbG.tvDDPt.uSP.ho6K Ti .xni v&s.zonuxzu
yx6yPxsTii t.zxDDPe.K T6.x.tSTPe.os.tPx6on.
youS.t T6Pt

.t Ti P.soK.yPxsTii t.T.uSP. C J l B
3 B(E) series:

6 non-contact seals (designation suffix
2RZ, 3j 1)

9 most common double row basic design
and SKF Explorer bearings:

6 shields (designation suffix 2Z, . 57x)

3 contact seals (designation suffix 2RS1,
. 57x 9)

For additional information, refer to *IBK26ud0*
v2d06, DbSgα .

When capped bearings must operate
under certain conditions, such as very high
speeds or high temperatures, grease may
appear between the inner ring and capping
device. For bearing arrangements where this
would be detrimental, appropriate actions
should be taken.

c

=bGh-5 bPh-Ci+Tthh+Ihh4
=hylnhP+HlC+GhNHH+Ch+Th+DnhGGUj

pn , pnd , dc i c

=TbKh+nC+PPWUcnb4#GNWcnb45 C5 hnl
=TbKh+Th+b5 h45 Uuj +tjhhPH+HCqhn=
chbGuj H
=iCG5 +n+ylGh5 h4+nbgGx j bq× WT+Th=
DnhGGUj +TCwPhG
=bGh-5 bPh-Ci+Tthh+Ihh4GhNHH+Ch+Th+DnhGGUj
bnP× hbG GhHbnlO
=5 b2hj CCP+qCHWU+ChNbn× WT+Th=
GhNHH+Ch+Th+DnhGGUj. uPe

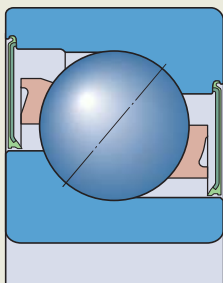
rpnnd , dc i c

.xsP.9 xeP.ohf Bj
.xsP.sPThosPe.K T6.x.tSPPut uPP6Tt Psu
.xsP. uPe.Ti.x.sPzPt.on.uSP.ovuPs.sTii .xne.
9 xnP.i ooe .Dot UWP.zonuxzuK T6.uSP.
sPzPt t
.SxwP.x.6D.uSxuPLPsu.6T SuDsPt tvsP.
xi xTt uSP.sPzPt.on.uSP.TnPs.sTii .u.
DsoWEP.xn.PHPzUWP.t Px6



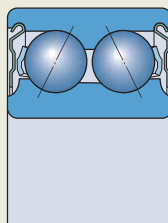
blj 1-

pbZZhP-chbsinj -6-nYnONynubN+hb6



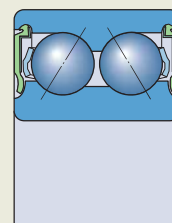
G0

pgr86 l 8gt n0 a2 8 6a



ulj 1-

pbZZhP-chbsinj -6-NynubN+hb6





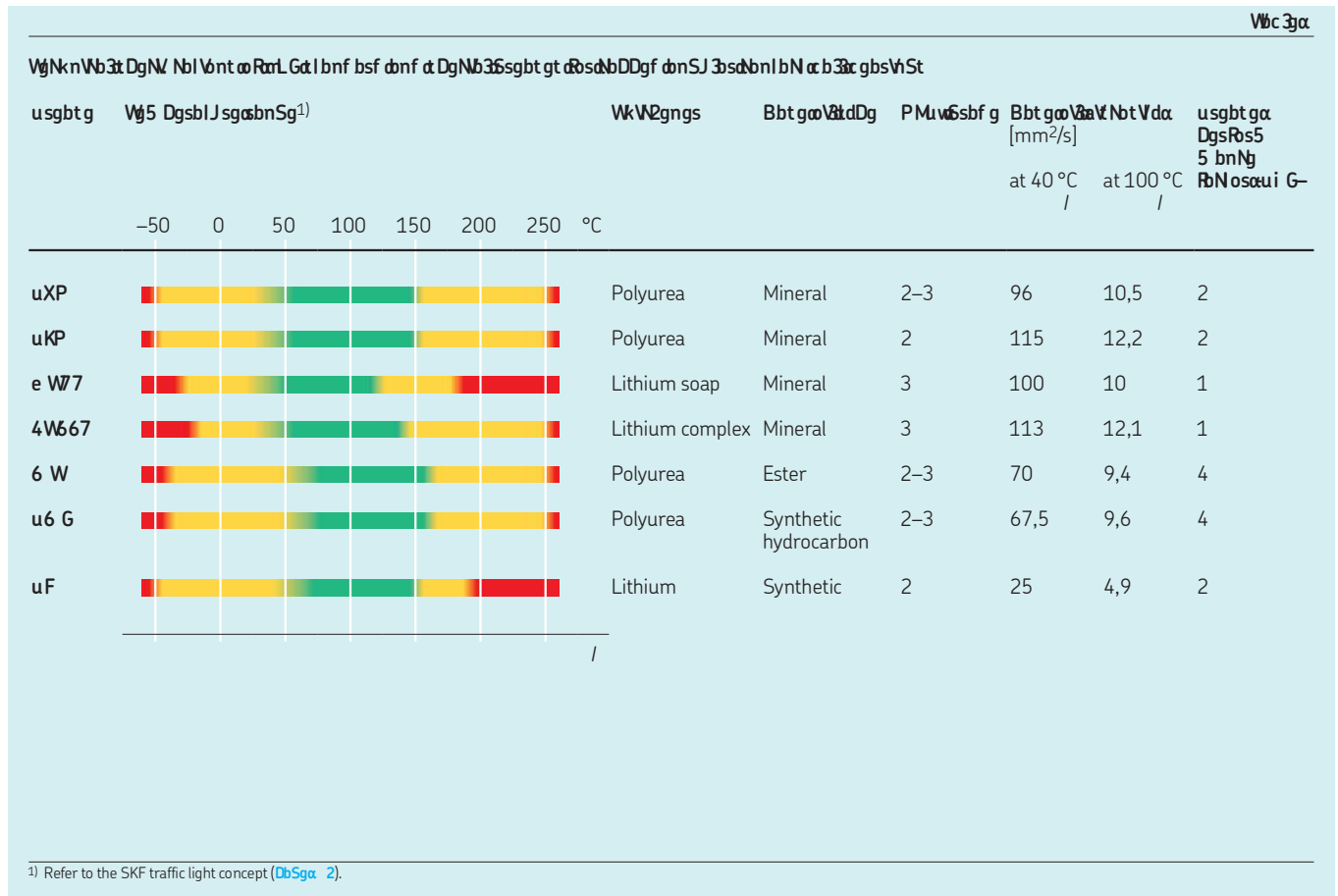
awwpuTFF- hfun w xt8ga8 -8-ot 3grr 86 l 8gt n0a

ohbGuj HnbqqhP=Cn=CIT+PhHbGh=4c Gbnlhp=
iCG#Th=4h=Ci#Th=hbGuj bnPbGhKQwb4e=
5 bdlhnbnN iGh VThdbCP. @Pe.K T5.onP.oh
uSP.hokK Thi .i sPxtPt. [tbl 8](#)

Grease life for capped angular contact ball bearings can be estimated as described for deep groove ball bearings ([DbSga](#)). The required grease information is provided in [lbc 3a](#).

- .t Thi @.soK.yPxsThi t.
- .xt.tuxnexse. .HXf
- .eovy@.soK.yPxsThi t
- .xt.tuxnexse. .Haf
- .Th.FvsoDP. .MT33 (commonly used and widely available)
- low-friction grease → GE2
- 9 other greases ([tbl 8](#)) can be supplied on request

The standard grease is not identified in the bearing designation (no designation suffix). Other greases are indicated by the corresponding grease suffix.



py l u

l Lt bnj w bGNCnlbN-εb4εhbGbj HbGP. uPe.
KTS.onP.os.uK o. eovyφ.sok.yPxsThi t .ohuSP.
zxi Pt.tSoKn.Th.d l .

l SP.tuxnexe.zxi Pt.oheovyφ.sok.yPxs
Thi t.xsP.PTSPs.9 xeP.ohRA pap cdT s
cd

T. n maTcc.T c nTdpn
tvh Ld .ohtThi φ.sok.yPxsThi t.Sxt.yPPn.
vDi sxePe.xt.hoKt



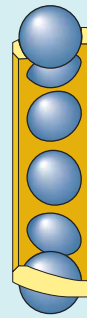
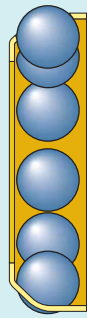
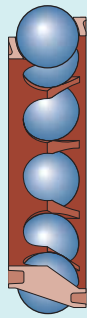
- . oDūT TPe.zxi P.DoznPui Po9 PūM
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- φxe.zonuPnu

: SPh.vtPe.xuST S.uP9 DPswsPt .to9 P.
φysExnut.zxn.SxwP.x.ePūēP Pnux6PttPzuon.
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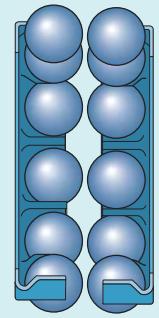
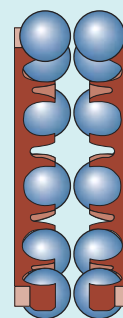
Zbj h-

pbj ht iYs-bnj v φs-NvubNucb6chbslnj t

rtnj φs-Yx -bnj v φs-NvubNucb6chbslnj t



EYvc φs-Yx -bnj v φs-NvubNucb6chbslnj t



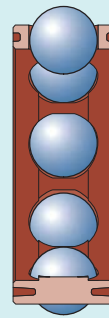
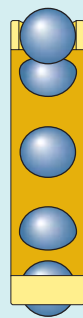
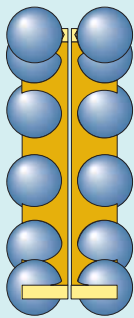
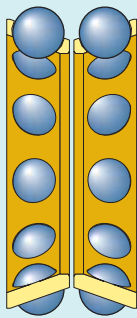
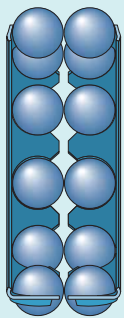
pbj h-udZh	: TheoK uMDP.yx6zPnusPe		: TheoK uMDP.. yx6zPnusPe		: TheoK uMDP.. yx6zPnusPe		kxuD uMDP.. yx6zPnusPe		kxuD uMDP.. yx6zPnusPe			
e buhslb6	RA-- i &tt. ysP. sPThoszPe		RFFb i &tt. ysP. sPThoszPe		kux9 DPe.ysxtt .tux9 DPe. tūPP6		d xzStiPe.ysxtt . 9 xzStiPe.tūPP6		RA-- i &tt. ysP. sPThoszPe		kux9 DPe.tūPP6	
mvi3y	R		RI		Y.a		d .G		lf •		a	

r lhl l) dGapaφ pa aB



il

bpea Fp nd,pnd ,dli l i a n c



<p>I nbq Idqhe= NBcx neeb4#hnlGhP</p>	<p>6 thPCx Idqhe= cb4#hnlGhP</p>	<p>SGCnj Idqhe= CwhGgbj #hnlGhP</p>	<p>6 thPCx Idqhe= CwhGgbj #hnlGhP</p>	<p>6 thPCx Idqhe= 4wc GbblWn# GCCKhHh#Th# wPshj #WGbN# CwhGgbj #hnlGhP</p>
<p>kux9 DPe.tuPP6</p>	<p>d xzSThPe.ysxtt</p>	<p>d xzSThPe.ysxtt</p>	<p>d xzSThPe.ysxtt</p>	<p>RFFb .i &tt. ysp.sPThroszPe</p>
	<p>d</p>	<p>d A</p>	<p>d A</p>	<p>RI Ak</p>

olys B Syw



n aph in eia, pnd, dli l ian c

t nc pn
cd n i a c

o CwnPbGd#P5 hnH0nH= | 2044

W03sbnNgt

Normal
Except for:
9 SKF Explorer bearings:
3 P6 dimensional tolerance
3 P5 geometrical tolerance
9 Bearings with D ≥ 400 mm:
3 P6 geometrical tolerance

For additional information → [DbSgα1](#)

Values: ISO 4 2 ([Ibc 3γα](#), [DbSgα 0](#) to [Ibc 3γα](#), [DbSgα 9](#))

ponl bN0bnS3γ

9 suffix B: 40°
9 suffix AC: 25°

For availability of bearings with 30° contact angle, contact SKF.

wl gsnb3N0ybsbnNg

mH S3γα gbsVhSt
Obtained after mounting, depending on adjustment against a second bearing.

i bVt ωRl nVgst b316 bIKbc 3γα gbsVhSt
9 CA – smaller than Normal axial clearance ([Ibc 3γα](#), [DbSgα](#))
9 CB – Normal axial clearance (standard) ([Ibc 3γα](#))
9 CC – larger than Normal axial clearance ([Ibc 3γα](#))
9 G (standard for larger bearings) – Normal axial clearance ([Ibc 3γd](#), [DbSgα](#))

For additional information → [DbSgα •](#)

Values are valid for unmounted bearing sets, arranged back-to-back or face-to-face under zero measuring load.

i sg3bf

mH S3γα gbsVhSt
Obtained after mounting, depending on adjustment against a second bearing.

i bVt ωRl nVgst b316 bIKbc 3γα gbsVhSt
9 GA – light preload (standard)
9 GB – moderate preload
9 GC – heavy preload

For additional information → [DbSgα •](#)

Values ([Ibc 3γα](#), [DbSgα 1](#)) apply to unmounted bearing sets, arranged back-to-back or face-to-face.

i gs5 VtV 3γα
5 Vb36n5 gnl

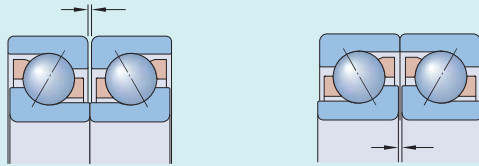
Back-to-back: ≈ 2 minutes of arc
Face-to-face: ≈ 4 minutes of arc

Misalignment increases bearing noise and reduces bearing service life, and when it exceeds ...



Gp ch – u0 f n c f u Tpnbf Tbhf h–f un w	vpcu Fp nb Tpnbf Tbhf h–f un w
Boundary dimensions: ISO 15 Except for: 9 bearing 3200 A: width = 14 mm instead of 14,3 mm 9 snap rings and grooves: ISO 464 (bPR 3 , DbSg4 1)	Boundary dimensions: ISO 15 Except for: 9 Locating slots: ISO 20515 (Ibc 3jα , DbSg4• 2)
Normal Except for: 9 SKF Explorer bearings and 33 DNRCBM series: 3 P6	Normal P6 geometrical tolerance on request Except for: 9 SKF Explorer bearings: 3 P6 3 width tolerance reduced to 0/–40 μm
9 32 A and 33 A series: 30° 9 33 D series: 45° 9 33 DNRCBM series: 40°	9 35°
Normal Check availability of C2, C3 or C4 clearance classes Values: (Ibc 3jα , DbSg4)	Normal Check availability of C2, C3, C4 or reduced ranges of standard clearance classes Values: ISO 5753-2 (Ibc 3jα , DbSg4 2)
Values are valid for unmounted bearings under zero measuring load.	
–	–
≈ 2 minutes of arc	≈ 2 minutes of arc
... the guideline values, these effects become particularly noticeable.	

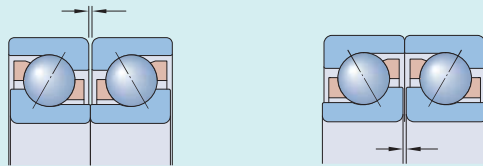
AyV3Wlgsnb3NgbsbnNgwRu nVagst b3i6 b lNkc 3a VhS3ox dnSJ 3sdnbl bN a b3c gbsVhSt dssbnSgf a bN25lo5c bN2asdbNg5lo5RbNg



= =

Bosgd Vb5 glgs		AyV3Wlgsnb3NgbsbnNg					
P		p3tt		Do		DD	
-		5 lth	5 by	5 lth	5 by	5 lth	5 by
5	5	5					
3	6)	9	7	9	7	8	7
6)	70	>	9	(:	7	8
70	90)	>		7	8	8(
90)0		7	:	7(8(:
)0	6 0	8	:	7	88	99	:>
6 0	6:0	>)	79	8>	:	>8
6:0	6)0	>)	79	8>	:	>8
6)0	90	:	7>	89	:	>8)
90	769	:=	8 =	9 =	:(=) =	:=

AyV3Wlgsnb3NgbsbnNgwRu d gt V6nd nVagst b3i6 b lNkc 3a VhS3ox dnSJ 3sdnbl bN a b3c gbsVhSt dssbnSgf a bN25lo5c bN2asdbNg5lo5RbNg

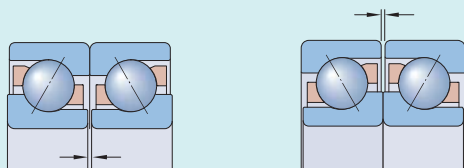


= =

Bosgd Vb5 glgs		AyV3Wlgsnb3NgbsbnNgwRu gbsVhSt dndkgdgsVt a													
P		> (A		>) A		> A		> e		> e		>7e		>8e	
-		5 lth	5 by	5 lth	5 by	5 lth	5 by	5 lth	5 by	5 lth	5 by	5 lth	5 by	5 lth	5 by
5	5	5													
70	:0														
:0	(0														
600	6:0														
6:0	80	9	:(9	:(9	:(>						
80)0	9	:(9	:(7	(7	(7	(
)0	700	9	:(9	:(7	(7	(7	(
700	780	9	:(7	(7	(7	(7	(8		
780	800	9	:(8		8		8		7	(:		
800	8 0	8		8		8		8		8			:		
8 0	8:0	8		8		8		8		:			:		
8:0	900	:		:		:		:		:			:		
900	(90	:		:		:		:		:			:		



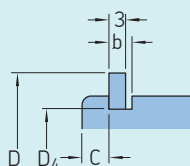
v3 CPmC5MB0BvaP S PyT9PR 3 a0B8 3 vCf PB8MPv TCBvPTy RP R3Pv0B8a PwPB83mRPT6 yC RPT6 Cv PPT3 yC PPT3



= =

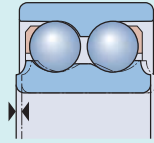
P	v3 CPm		u0		uD	
	uA 5 l4	5 by	5 l4	5 by	5 l4	5 by
5 5	5		5		5	
	18	8			(:
	18	8			(:
	18	8			(:
	1:	:	7	9		8
	1:	:	7	9		8
	1:	:	7	9		8
	1((8		:	7
	1(=	(=	8 =	=	: =	7 =

t 0 3Ba0CBa C5aBPr v0B8 8vCCN3a PBmaBPr v0B8a



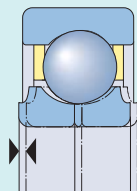
o3Pv0B8 r hHj nbl 0n	t 0 3Ba0CBa					1BPr v0B8 r hHj nbl 0n
=	D	c	i	r7	r8	=
	5 5					
t 2 - Dok	7• (•	•	(: {) : •	1
t 2 - Dok	7• (•	•	•) : {	•) : •	1
t 2 - Dok	7• (•	•	•) : {	•) : •	1
t 2 - Dok	8• :	7•	•	9•) •	1
t 2 - Dok	8•)	7•	•	79•	8•) •	1

Aylb6lnuhsnb6N8hsbnNn-Yi-PYvc 6-sYx -bnj v 6s-NVnubNuc b6c hbslnj t



oYsh-Plb8 huhs	Aylb6lnuhsnb6N8hsbnNn-Yi-c hbslnj t ln-ukh-t hslht								((-E		((-Eg Upoe		
P	D		g CG5 b4		D7		D8		= 5 5		= 5 5		
-	5 5	5 by	5 5	5 by	5 5	5 by	5 5	5 by	5 5	5 by	5 5	5 by	
5 5	5								5		5		
6			9		(9	89	9	89				
.	:)	8	>	9	:	78	(8(>	8>	:	:	
:)	(9	(>	(7>	7	9	7	9	:	:	
()	:))	8	77	98	77	98			7	
)	fi	(77	7	88	7:	9(7:	9(7	
fi	, fi	7		7	7:	:	8(8	: 7	8	: 7	(7(
, fi	.	7	8	9	8	7	98	8:	>	8:	>	(7(
.		7	:	(8:	79	: 7	99	(7	99	(7		
		8	7		97	8	>7	: 9):	: 9):		

Ai g nb8tng 38gtgn38 o--oct ro nb3onhg3bl g l 8gt n0a



Bot8 6 g 8b8t	Ai g nb8tng 38gtgn38		g CG5 b4		D7		D8	
P	5 uth	5 by	5 uth	5 by	5 uth	5 by	5 uth	5 by
5 5	5							
	9	: 9	9) 9	(9	7		: 9
	9	>9	: 9			9	79	(9
	79	(9	>9	9		: 9	9	
	89		(9	8	9	>9	: 9	9
	99) 9	9	79)	(79
	>	7	9	>9	:		9	: 9
)	99	79		(9	9	79	7
	9	>9	99	9	(:	77
) 9	>9	9	7	7 9)	7:
	79	9) 9	>9	99	779	7 9	7)
	99	8	(9	7 9	(9	7>	79	87
	>9	: 9	89	77	7	8	7(8>
)	(9	: 9	7:	78	879	8 9	9
		7)	7)	7: 9	8>	889	989



f CySu



	n aph in eia,pnd,dli l ian c	t pel aph in eia,pnd,dli l ian c
- n e pi	<p>e Inl8 v8 -bylb64/bP-iYs-tlnj 6h-chbslnj t -bnP-c-hbslnj - Zblst -bssbnj hP-ln-ubnPh8 0</p> $F_{am} = A \left(\frac{n}{1000} \right)^2$	
t CGbPPWUcnb4 lhiCG5 blUcn= Zbj h-	<p>e VhV5 J5 asbf V33bf asbf asbf gbsVhSdbVt dssbnSgf α c bN25l o5c bN2as bN5l o5RbNj,</p> $F_{rm} = k_r \left(\frac{v n}{1000} \right)^{2/3} \left(\frac{d_m}{100} \right)^2$	<p>e VhV5 J5 asbf V33bf ,</p> $F_{rm} = k_r \left(\frac{v n}{1000} \right)^{2/3} \left(\frac{d_m}{100} \right)^2$
Fr J Vab3ynl α f dnb5 Vx gbsVhSα 3bbf	<p>mVhS3α gbsVhSt dnf α gbsVhSdbVt dssbnSgf dVhα lbnf g5 ,</p> $F_a/F_r \leq e \rightarrow P = F_r$ $F_a/F_r > e \rightarrow P = X F_r + Y_2 F_a$ <p>When determining the axial load F_a, refer to <i>pd0b@k06 K 2 d0@l00dl 4Cuf 2du06v 1 CbBk2l v060 Cur d02l 0 KlBl 21 DbSgα 99.</i></p> <p>BgbsVhSdbVt dssbnSgf α bN25l o5c bN2asα RbN5l o5RbNj,</p> $F_a/F_r \leq e \rightarrow P = F_r + Y_1 F_a$ $F_a/F_r > e \rightarrow P = X F_r + Y_2 F_a$	$F_a/F_r \leq e \rightarrow P = F_r + Y_1 F_a$ $F_a/F_r > e \rightarrow P = X F_r + Y_2 F_a$
For additional information → DbSgα		
Fr J Vab3ynl α l b l Vx c gbsVhSdbbf	<p>mVhS3α gbsVhSt dnf α gbsVhSdbVt dssbnSgf dVhα lbnf g5 ,</p> $P_0 = 0,5 F_r + Y_0 F_a$ $P_0 < F_r \rightarrow P_0 = F_r$ <p>When determining the axial load F_a, refer to <i>pd0b@k06 K 2 d0@l00dl 4Cuf 2du06v 1 CbBk2l v060 Cur d02l 0 KlBl 21 , DbSgα 99.</i></p> <p>BgbsVhSdbVt dssbnSgf α bN25l o5c bN2asα RbN5l o5RbNj,</p> $P_0 = F_r + Y_0 F_a$	$P_0 = F_r + Y_0 F_a$
For additional information → DbSgα 91		



vpcu Fp nb Tpnlf Tbhf h-f un w	
<p>e ln18 v8 -bylb66/bP0</p> $F_{am} = A \left(\frac{n}{1000} \right)^2$	<p>md8 cY6</p> <p>A= 5 0 5 w5 -bylb44CbP+bNCG=ZsYPvNubc6nt0 = luj 4-Cx €hbGuj H-DbSgα 9 = CwG qCul+CNlbN €hbGuj H-DbSgα 49</p> <p>P₅ chbGuj -5 hbn+P05 hlhGp5 5 = 0,5 (d + D)</p>
-	<p>e calculation factor for single and double row bearings (lbc 3α 9, DbSgα 99)</p> <p>F_a axial load [kN] F_{am} minimum axial load [kN] F_r radial load [kN] F_{rm} minimum radial load [kN] k_r minimum radial load factor (Dsof J N dbc 3jt) 9 Single row bearings, DbSgα 9 9 Double row bearings, DbSgα</p>
<p>Locating bearings to accommodate radial and axial load: $F_a/F_r \leq 0,5 \rightarrow P = F_r + 0,66 F_a$ $F_a/F_r > 0,5 \rightarrow P = 0,6 F_r + 1,07 F_a$</p> <p>For a proper functionality, SKF recommends $F_a \geq 1,27 F_r$.</p> <p>Thrust bearings with radial clearance in the housing in combination with a radial bearing (. S7α , DbSgα 94): $P = 1,07 F_a$</p>	<p>n rotational speed [r/min] P equivalent dynamic bearing load [kN] P₀ equivalent static bearing load [kN] X, Y₀, Y₁, Y₂ calculation factors for single and double row bearings (lbc 3α 9) ν actual operating viscosity of the lubricant [mm²/s]</p>
$P_0 = F_r + 0,58 F_a$	

py gK ywB wl ydy CyS Cs el ys B u CKBw S uB PCs Dy sl S B wyBSI

When a radial load is applied to a single row angular contact ball bearing, the load is transmitted from one raceway to the other at an angle to the bearing axis and an internal axial load is induced. This must be considered when calculating the equivalent bearing loads for bearings in adjusted arrangements consisting of two single bearings and/or bearing pairs arranged in tandem.

The equations (1bc 3α) are only valid if the bearings have identical contact angles and are adjusted against each other to practically zero clearance, but without any preload. In the table, bearing A is subjected to a radial load F_{rA} and bearing B to a radial load F_{rB} . Both F_{rA} and F_{rB} are always considered positive, even when they act in the direction opposite to that shown in the figures. The radial loads act at the pressure centres of the bearings (distance a, refer to [Dsof J N dbc 3gt, DbSgα 9](#)).

These calculations can easily be done with SKF's online calculation tools. When the bearings are adjusted with clearance or preload, or when bearings with different contact angles are used, the equations become more complex and can be done using the SKF SimPro platform (skf.com/simpro).

f CyS gyssPB gyDyg vPC el ys B Dy su

WTh#b4wh#CGεbHN#CbP-βlWj HbnP#blWjwh=
CbP-45 W#H#hP#h#Th **Fap e, d d l c Fi**
9 εbqq4#C#Hj 4εhbGhj HεCGεhbGhj =
qblW-5 CwnlhP-45 5 hPblh4εbPbnhl#C#hbNT=
ClThG#Th#C4Cx Wj #b4wh#bqq4,

- =c bHN#Pdnb5 W#CbP-βlWj #CG#HbnPbP=
- c hbGhj H#b4bGbnj h5 hnlHbnP#CG# Lt =
- syq4CGεhbGhj H#b4εbN2 IC cbN2-εG=
- ibN# IC ibN#bGbnj h5 hnl
- D = 1,62 C_{single bearing}
- 9 basic dynamic load rating for SKF Explorer bearings in a tandem arrangement
- C = 2 C_{single bearing}
- 9 basic static load rating
- C₀ = 2 C_{0 single bearing}
- 9 fatigue load limit
- P_u = 2 P_{u single bearing}



Wbc 3α 9

pb3V 3el Vndbn ost dssd VhS3gnf ε oJ c 3sox dnSJ 3sdbnl bN α b3ε gbsVhSt

BgbsVhSα ldDgt	pb3V 3el Vndbn os e X	Y ₁	Y ₂	Y ₀
mH S3sox α gbsVhSt				
mH S3α gbsVhSt αsα gbsVhSα				
Db'lt dssbnSgf dH dbnf g5				
mJ R ydB	1,4	0,35	–	0,57
mJ R yAp	0,68	0,41	–	0,87
BgbsVhS Db'lt dssbnSgf α bN25				
lo5c bN2 αsdbNg5 lo5RbNg				
mJ R ydB	1,14	0,57	0,55	0,93
mJ R yAp	0,68	0,67	0,92	1,41
EoJ c 3sox α gbsVhSt				
mgsVt α 7d 7d				
mgsVt α 7dE	1,34	0,54	0,47	0,81
mgsVt α 7dE Pl pBe	1,14	0,57	0,55	0,93

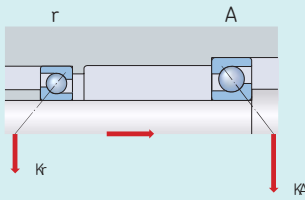
Aylb66/bPtInj -Yi-c hbslnj -bssbnj h8 hnut -InNysZYsbuInj -uk Y-tInj 6-sYx -bnj v 6s-NWnubNk-cb6c hbslnj t -bnP Ys-c hbslnj -Zblst -In-ubnPh8

o hbslnj -bssbnj h8 hnu

M/bP-Nbth

Aylb66/bPt

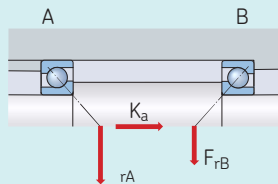
o bN4Ou/OcbN4



pbth- b

$$t_{GA} \quad g_{eo} \quad g_{(A)} \quad g_{eA} \quad g_{(o)} \quad g_{(A)}$$

GbN4Ou/OibN4



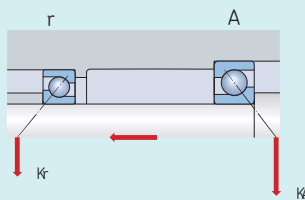
pbth- c

$$g_{eA} \quad g_{eo} \quad g_{(A)} \quad g_{eA} \quad g_{(o)} \quad g_{(A)}$$

pbth- N

$$g_{eA} \quad g_{eo} \quad g_{(A)} \quad g_{(o)} \quad g_{eo}$$

o bN4Ou/OcbN4

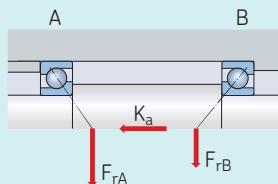


pbth- b

$$g_{eA} = t_{G0} \quad t_{bA} \leq t_{bo} \leq t_b \quad t_{bo} \leq t_{G0}$$

Lb

GbN4Ou/OibN4



pbth- c

$$g_{eA} \quad g_{eo} \quad g_{(A)} \quad g_{(o)} \quad g_{eo}$$

pbth- N

$$g_{eA} \quad g_{eo} \quad g_{(A)} \quad g_{eA} \quad g_{(o)} \quad g_{(A)}$$

gre- (en h > i
 20° contact angle → R = 0,50
 25° contact angle → R = 0,57
 30° contact angle → R = 0,66
 40° contact angle → R = 0,88



l DlsyKsl vu

The permissible operating temperature for angular contact ball bearings can be limited by:



- 9 the dimensional stability of the bearing rings and balls
- 9 the cage
- 9 the seals
- 9 the lubricant

Where temperatures outside the permissible range are expected, contact SKF.

o hbslnj -slnj t -bnP-c b&

The bearings are heat stabilized up to at least 150 °C 4 1u2

pbj ht

Steel, brass or PEEK cages can be used at the same operating temperatures as the bearing rings and balls. For temperature limits of cages made of other polymer materials, refer to) r 30 t Se+ a, Zbj h-

mhb&

The permissible operating temperature for NBR seals is -40 to +100 °C 4 k 1u2 Temperatures up to 120 °C 4 1u2 can be tolerated for brief periods. Typically, temperature peaks are at the seal lip.

M csl Nbnut

Temperature limits for greases used in sealed SKF angular contact ball bearings are provided in ujl 8 , DbSg&. For temperature limits of other SKF greases, refer to - 3 SKb+e 1b %eg 3 1-Pu 1t ea , DbSg& . When using lubricants not supplied by SKF, temperature limits should be evaluated according to the SKF traffic light concept (DbSg& 2).

-l s uuel uDl l S

With the help of the following formula, the permissible operating speed can be determined,

$$v = \frac{d_m \cdot n}{1000} \cdot c_F \cdot c_T \cdot c_H$$

where:

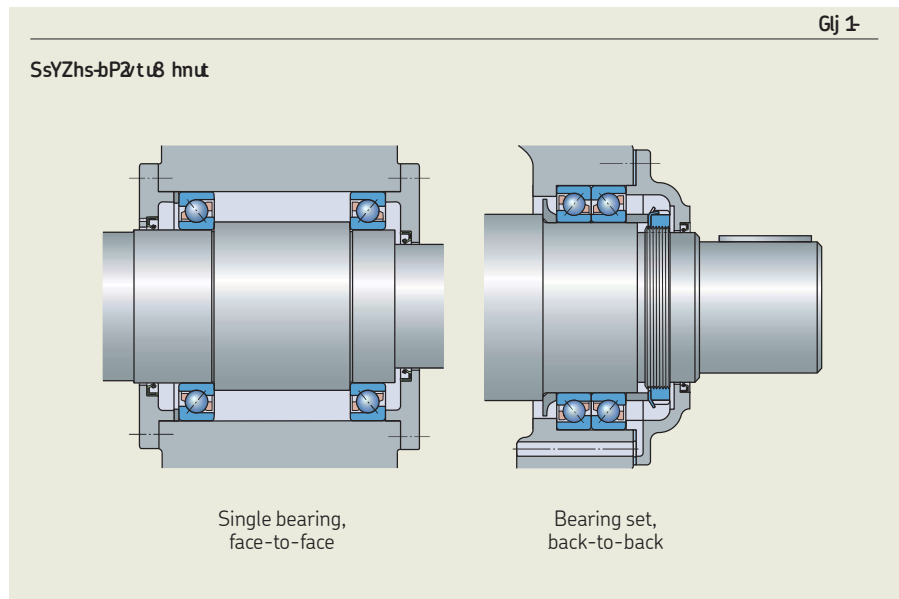
- d_m = bearing mean diameter [mm]
- n = rotational speed [r/min]
- c_F = bearing factor
- c_T = temperature factor
- c_H = housing factor

CGb PPUWnb 4hi CG5 bl Wn e Gh G+ C 3 r 2ud K 06
k21 r 2ud k u2 dBl vr 22l e Zbj h-
l Lt Gh NC5 5 hn PH C U 4 w G W bl Wn + CG e hb G
Uj Hx W T b G Uj hnl Gh P + bj h = Ph H j nbl Wn =
tvh L d A. os. RI Ak : S Ph. u S Pt P. y P x s T i t. x s P.
i s P x t P. 6 y s X u P e . u S P. ne 9 . v x 6 P. T. 6 T P e. u.
250 000 mm/min.

where
 d_m = bearing mean diameter [mm]
 $= 0,5 (d + D)$
 n = rotational speed [r/min]

o hbslnj -Zblst

For bearings arranged in pairs, the limiting speed should be reduced to approximately 80% of the value quoted for a single bearing.



El u B gCBu Sl sywCBu

B l sCL yB Kys gCBwygwey elys B u

upF-uf cwb -nb

l luj 4n-εCx #nj w4GACnlbNεb4εhbGtj H-
5 wHεh#wHhP=3j 1 Q=

=x WT #hNCnPεhbGtj
=h+hIH

WThεhbGtj H5 wHεh#PmHhP#j bblH=
hbNT-εlThGwnlU#Th-εFwWWh-NhbGbnNεCG-
qGh4CbP-#CclblhP=(2Q; K86 ru20dl =
Zbj h-

3nWkGh4d-5 blNTbc 4nεhbGtj H5 CwnlhP=
U 5 hPlh4d#PmHhNhl#C#bNT-εlThG

=GhFwGh#C#wG ThG#PmH5 hnl=0 2du86v
4CubB92ud01 dK 86#Zbj h- 0
=Cc l btl#GhFwWWh-NhbGbnNεCGεGh4CbPεd,
6=NTCChtj εhbGtj H#GC5 #n#bqCqGblh=
NhbGbnNεCGεGh4CbP-NbHh
6=bqC4luj #Wlxyφ. ut.hos.εSP.yPxsThi t.on.
εSP.tSxlhxne.Th.εSP.Sovt Thi

RPshs9 xnzP.xne.oDPsxuBnx6sP6ky BEM
ePDPne.on

. DsoDPs.xelWt u9 Phulost Thi φ.yPxsThi t
. εSP.zossPzut PφzuBn.ohzφxsxnzP.xne.
DsP6xe.hos.vnWPstxφM9 xuzSxyφ.yPxsThi t
.huSPsP.T.ωo.9 vzS.zφxsxnzP.Th.εSP.
yPxsThi .xssxni P9 PhuevsThi .oDPsxuBn .
εSP.6xe.zxssMThi .zxDxzTMohuεSP.yPxsThi t.
K 6nouyP.lv φAvUBPe .FLzPt t WP.DsP6xe.
DsoevzPt.9 osP.hsZuBn.xne.ST SPs.oDPsxu
Thi .LP9 DPsxuvsPt .φxeThi .u.x.sPevzuBn.Th.
yPxsThi .tPswεP.φP

Aylb6d/bPt -In-Ynh-PishNlYn

: SPn.εSP.xLk66xe.xzut.DsPeo9 ThxnuMTh.
onP.eεPzuBn.Th.yxzmω yxzmεne.lxzP ω
lxzP.xssxni P9 Pnut .vnlxwvxyφ.soφni .
zone Tdbnt.hos.εSP.yxφ.ohuεSP.xLk66
vn6xePe.yPxsThi .9 xMozzvs.KSεS.zxn.φxe.
ω

. ThzsPxt Pe.noTEP.φvP6
. eTzonuThv TMTh.εSP.φysεXnu φ
. ThzsPxt Pe.tuPt t Pt.on.εSP.zxi P

WhePs.εSP.Pzεzv9 tuxnzPt .kbGsPzo9
9 Pnet.NPso.oDPsxuThi .zφxsxnzP.KSεS.zxn.
yP.xuxThPe.yMvt Thi .tDsThi t.: SPn.tDsThi t.
xsp.noutvh zPnu.vt Thi .yPxsThi t.KTεS.25°
contact angle as a backup bearing may help.

MwbP-sbuY

- 9 of $F_a/F_r \geq 1$ is required by bearings in the 70 B, 72 B(E) and 73 B(E) series
- 9 of $F_a/F_r \geq 0,55$ is required by bearings in the 72 AC and 73 AC series

If the load ratio requirement is not met in each case, bearing service life can be reduced.

tCKs DC BwgCBwygw ey elys B u

5thP-bt -b-ksvt ucbslnj

Four-point contact ball bearings are often used as entirely thrust bearings, together with a radial bearing. When used in this way, the four-point contact ball bearing should be mounted with radial clearance in the housing (3j 1).

9 in combination with a cylindrical roller bearing:

the radial internal clearance of the cylindrical roller bearing should be smaller than the theoretical radial internal clearance of the four-point contact ball bearing after both have been mounted the theoretical radial clearance can be calculated from:

$$C_r = 0,7 C_a$$

where

C_r = theoretical radial internal clearance
 C_a = axial internal clearance (tbl 8 ,
DbSg6 2)

3 the outer ring of the four-point contact ball bearing must be able to accommodate thermal movements

Therefore, it should not be clamped axially, but a small gap should be maintained between the outer ring and the cover flange.

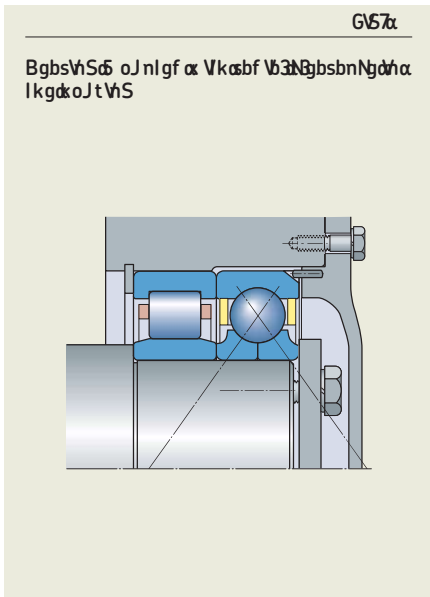
3 bearings with locating slots should be used (. 57x) to prevent the outer ring from turning

If clamping the outer ring cannot be avoided, the outer ring must be carefully centred during mounting.

Mbf εbl V

For proper functionality, the balls should contact only one inner ring raceway and the opposite side of the outer ring raceway. This is the case when the load ratio is $F_a/F_r \geq 1,27$.

A load ratio that is smaller than recommended can reduce bearing service life.



El u BywCB uPuw



ah3yht

obtINPhtlj nbuYn

Listed in **u bc 6-), Zbj h-**

AMm Inch bearing

Ae m Inch bearing

mvi3yht

HsYvZ- 0:huhsnb6Phtlj n

- A** Single row bearing, 30° contact angle
- A** Double row bearing, no filling slots
- Ao** Single row inch bearing, 20° contact angle
- Ap** Single row bearing, 25° contact angle
- o** Single row bearing, 40° contact angle
- E** Two-piece inner ring
- F** Optimized internal design

HsYvZ: 0Fyuhsnb6Phtlj n-1hb&7t nbZ-slnj-j sYYwh7hyhNv uYn7huN5

- g** Snap ring groove in the outer ring
- gU** Snap ring groove in the outer ring, with appropriate snap ring
- g** One locating slot (notch) in one outer ring side face
- g:** Two locating slots (notches) in one outer ring side face, 180° apart
- po** Double row bearing, controlled axial internal clearance
- pA** Bearing for universal matching. Two bearings arranged back-to-back or face-to-face have axial internal clearance smaller than Normal (CB).
- po** Bearing for universal matching. Two bearings arranged back-to-back or face-to-face have Normal axial internal clearance.
- pp** Bearing for universal matching. Two bearings arranged back-to-back or face-to-face have axial internal clearance greater than Normal (CB).
- H** Bearing for universal matching. Two bearings arranged back-to-back or face-to-face have axial internal clearance.
- HA** Bearing for universal matching. Two bearings arranged back-to-back or face-to-face have light preload.
- Ho** Bearing for universal matching. Two bearings arranged back-to-back or face-to-face have moderate preload.
- Hp** Bearing for universal matching. Two bearings arranged back-to-back or face-to-face have heavy preload.
- O: Um** Contact seal, NBR, on both sides
- O: UD** Non-contact seal, NBR, on both sides
- O: D** Shield on both sides

HsYvZ-(0pbj h-Phtlj n

- 6** Stamped steel cage, ball centred (double row bearing)
- G** Machined steel cage, ball centred
- GA** Machined steel cage, outer ring centred
- a** Stamped steel cage, ball centred (single row bearing)
- a** Stamped steel cage, ball centred (double row bearing with a two-piece inner ring)
- e** Machined brass cage, ball centred; different designs are identified by a number following the M, e.g. M2
- e A** Machined brass cage, outer ring centred.
- e o** Machined brass cage, inner ring centred
- S** Glass fibre reinforced PA66 cage, ball centred
- SI** Glass fibre reinforced PEEK cage, ball centred
- SI Am** Glass fibre reinforced PEEK cage, with lubrication grooves in the guiding surfaces, outer ring centred
- 3g/** Glass fibre reinforced PA66 cage, ball centred
- C** Stamped brass cage, ball centred